

# **SURVEY OF GREEN'S FUNCTION RESEARCH RELATED TO TRANSIENT HEAT CONDUCTION**

**James V. Beck, Prof. Emeritus, Mech. Eng., Michigan State University, E. Lansing, MI and Beck Engineering Consultants Co., Okemos, MI. ([www.BeckEng.com](http://www.BeckEng.com))**

**Other team members include: Kevin Cole, Bob McMasters, David Yen, A. Haji-Sheikh, Don Amos**

**Research supported by Sandia National Labs, Albuquerque, NM. Kevin Dowding, Project manager**

# **MOTIVATION**

## **UNIQUE OPPORTUNITIES NOW**

**Exploding knowledge**

**“Infinite” computer memory**

**World-wide internet connectivity**

## **APPLICATIONS**

**Education**

**Retention/availability of unique contributions**

**Verification**

# THREE-DIMENSIONAL HEAT CONDUCTION EQUATION IN CARTESIAN COORDINATES

$$k \left[ \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right] + g = C \frac{\partial T}{\partial t}$$

**Domain:  $0 < x < L, 0 < y < W, 0 < z < H$**

**$T$  = Temperature**

**$k$  = Thermal conductivity**

**$g$  = Volumetric Energy Generation**

**$C$  = Volumetric Heat Capacity**

**$\alpha = k/C =$  Thermal Diffusivity**

## Boundary Conditions:

$T(0, y, z, t) = T_0$ , Prescribed  $T$ , 1<sup>st</sup> kind

$k \frac{\partial T(L, y, z, t)}{\partial x} = q_0$ , Prescribed heat flux, 2<sup>nd</sup> kind

$-k \frac{\partial T(x, 0, z, t)}{\partial y} = h(T_\infty - T(x, 0, z, t))$ , Convective, 3<sup>rd</sup> kind

If no physical boundary, such as for  $x \rightarrow \infty$  or  $r$  in radial coordinates  $\rightarrow$ , 0<sup>th</sup> kind.

## GREEN'S FUNCTION SOLUTION EQUATION

For homogeneous,  $T$ -independent properties:

$$T(\mathbf{r}, t) = T_{in}(\mathbf{r}, t) + T_g(\mathbf{r}, t) + T_{b.c.}(\mathbf{r}, t)$$

where the initial condition term is

$$T_{in}(\mathbf{r}, t) = \int_V G(\mathbf{r}, t | \mathbf{r}', 0) F(\mathbf{r}') dv'$$

For volumetric energy generation,

$$T_g(\mathbf{r}, t) = \frac{\alpha}{k} \int_{\tau=0}^t \int_V G(\mathbf{r}, t | \mathbf{r}', \tau) g(\mathbf{r}', \tau) dv' d\tau$$

For the nonhomogeneous boundary conditions

$$T_{b.c.}(\mathbf{r}, t) = \frac{\alpha}{k} \int_{\tau=0}^t \sum_{i=1}^s \int_{A_i} G(\mathbf{r}, t | \mathbf{r}_i', \tau) f_i(\mathbf{r}_i', \tau) ds_i' d\tau$$

$$+ \alpha \int_{\tau=0}^t \sum_{j=1}^s \int_{A_j} \left( - \frac{\partial G(\mathbf{r}, t | \mathbf{r}_j', \tau)}{\partial n'} \Big|_{\mathbf{r}=\mathbf{r}_j'} \right) f_j(\mathbf{r}_j', \tau) ds_j' d\tau$$

where the first line is for b.c. of 2<sup>nd</sup> & 3<sup>rd</sup> kinds and second line is for b.c. of 1<sup>st</sup> kind.

For homogeneous rectangle or parallelepiped,  $G(\mathbf{r}, t | \mathbf{r}', \tau)$  can be written as a product of 1D GFs. For 3D case,

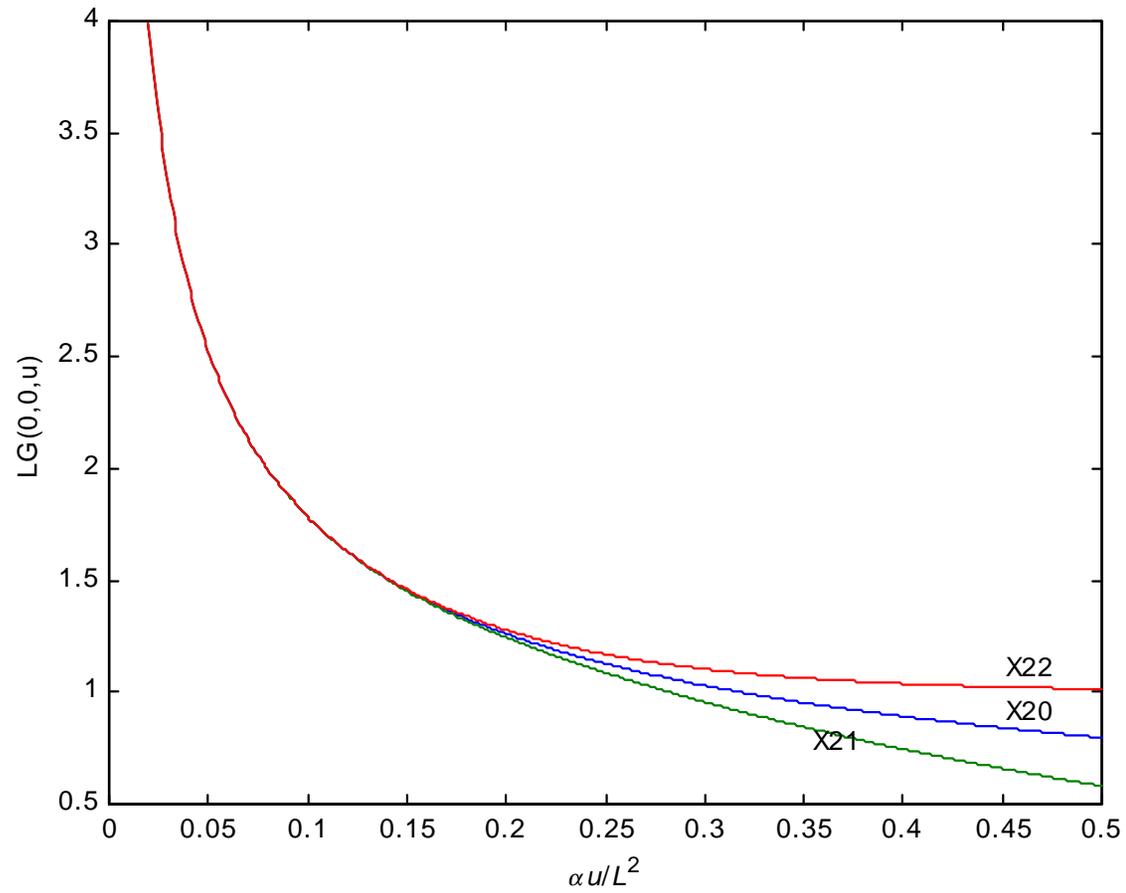
$$G(\mathbf{r}, t | \mathbf{r}', \tau) = G_X(x, t | x', \tau) G_Y(y, t | y', \tau) G_Z(z, t | z', \tau)$$

**Consider a boundary condition of 2<sup>nd</sup> kind at  $x = 0$  (a constant heat flux,  $q_0$ ) and boundary conditions of the 1<sup>st</sup> kind at all the other 5 boundaries. Then**

$$G(\mathbf{r}, t | \mathbf{r}', \tau) = G_{X21}(x, t | x', \tau) G_{Y11}(y, t | y', \tau) G_{Z11}(z, t | z', \tau)$$

$$T_{b.c.}(x, y, z, t) =$$

$$\frac{\alpha}{k} q_0 \int_{\tau=0}^t G_{X21}(x, t | 0, \tau) \int_{y'=0}^W G_{Y11}(y, t | y', \tau) dy' \int_{z'=0}^H G_{Z11}(z, t | z', \tau) dz' d\tau$$



$LG_{X20}(0,0,u)$ ,  $LG_{X21}(0,0,u)$ , and  $LG_{X22}(0,0,u)$  vs. dimensionless time;  $u \equiv t - \tau$

$\alpha u / L^2$	$LG_{X20}(0,0,u)$	$LG_{X21}(0,0,u)$	$LG_{X22}(0,0,u)$
0.040	2.820947918	2.820947918	2.820947918
0.050	2.523132522	2.523132512	2.523132532
0.060	2.303294330	2.303294064	2.303294596
0.070	2.132436186	2.132433521	2.132438851

$$LG_{X20}(0,0,u) = (\pi\alpha u / L^2)^{-1/2}$$

$$LG_{X21}(0,0,u) = 2 \sum_{m=1}^{\infty} e^{-\left(\frac{(2m-1)\pi}{2}\right)^2 \frac{\alpha u}{L^2}}$$

$$LG_{X22}(0,0,u) = 1 + 2 \sum_{m=1}^{\infty} e^{-(m\pi)^2 \frac{\alpha u}{L^2}}$$

## OBSERVATIONS

1. Large part of GF at short (i.e., recent) times.

- 2. Accurate & simple approximation for short times.**
- 3. For long time GF, want  $\exp[-(m_{max}\pi)^2 \alpha u / L^2]$  to be small. Note  $\exp(-2\pi^2) \approx 3E-9$ . Then  $m_{max} = L(2/\alpha u)^{1/2}$**
- 4. Fewer terms in long time GF for small  $L$  and large  $u$ .**

## **WAYS TO IMPROVE EFFICIENCY AND ACCURACY**

- A. Use short and long time GF. Time partitioning.**
- B. Use maximum  $t$  possible in long time GF.**
- C. For long time GF and small  $t$ , use artificially small  $L$ . Spatial partitioning.**

## NUMBERING SYSTEM

**MOTIVATION-Many geometries and boundary conditions**

**EXAMPLE: Temperature b.c. = 1<sup>st</sup> kind**  
**Heat flux b.c. = 2<sup>nd</sup> kind**  
**Convective b.c. = 3<sup>rd</sup> kind**  
**No physical boundary = 0<sup>th</sup> kind**

**In heat conduction, one b.c. at each boundary.**

***X* for *x*-direction, *Y* for *y*-direction, *Z* for *z*-direction**

***XIJ* is for plate with *I*th b.c. at  $x = 0$ , and *J*th b.c. at  $L$**

**Suppose  $T$  given at  $x = 0$  & convection b.c. at  $L$ : X13**

**Different possibilities in  $x$ -coordinates:**

**X00**

**X10   X11   X12   X13**

**X20   X21   X22   X23**

**X30   X31   X32   X33**

**A Green's function can be given for each of these.**

**Except for of 0<sup>th</sup> kind, we give TWO forms of each.**

**They are complementary in that one is more efficient than the other in different time domains.**

They can be used to provide internal verification.

## EXAMPLE X11 GF

Form best for small  $t - \tau$ ,

$$G_{X11}(x, t | x', \tau) = \frac{1}{\sqrt{4\pi\alpha(t-\tau)}} \sum_{n=-\infty}^{\infty} \left[ e^{-\frac{(2nL+x-x')^2}{4\alpha(t-\tau)}} - e^{-\frac{(2nL+x+x')^2}{4\alpha(t-\tau)}} \right]$$

Form best for large  $t - \tau$ ,

$$G_{X11}(x, t | x', \tau) = \frac{2}{L} \sum_{m=1}^{\infty} e^{-\beta_m \frac{\alpha(t-\tau)}{L^2}} \sin\left(\beta_m \frac{x}{L}\right) \sin\left(\beta_m \frac{x'}{L}\right) \quad \beta_m = m\pi$$

By selecting  $\alpha(t-\tau)/L^2 \approx 0.05$ , only a few terms needed.  
Related to our method of “time partitioning”

**Also some functions of GFs are convenient to have:**

$$-\frac{\partial G_{x11}}{\partial n'} \Big|_{n'=0}, \quad -\frac{\partial^2 G_{x11}}{\partial x \partial n'} \Big|_{n'=0}, \quad \int_{x'=0}^L G_{x11} dx', \quad \int_{x'=0}^L \frac{\partial G_{x11}}{\partial x} dx'$$

**For 3D problems in Cartesian coordinates,  $G = G_x G_y G_z$**

## **CYLINDRICAL RADIAL & RADIAL/ANGULAR**

**Many of these are also tabulated in our book, “Heat Conduction Using Green’s Functions” by J.V. Beck, K.J. Cole, A. Haji-Sheikh and B. Litkouhi**

**In the book, radial spherical Green’s functions are given.**

## CONDUCTION WITH SOLID BODY FLOW

**Consider the differential equation:**

$$k \left[ \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right] + g = C \left[ \frac{\partial T}{\partial t} + U \frac{\partial T}{\partial x} + V \frac{\partial T}{\partial y} + W \frac{\partial T}{\partial z} \right]$$

**where  $U$ ,  $V$  and  $W$  are constants and velocities in the  $x$ ,  $y$  and  $z$  directions, resp.**

**The same boundary conditions as used above are used.**

**Flow eq. can be transformed to heat conduction using:**

$$T(x, y, z, t) = W^*(x, y, z, t) \exp \left[ \frac{Ux}{2\alpha} - \frac{U^2 t}{4\alpha} + \frac{Vy}{2\alpha} - \frac{V^2 t}{4\alpha} + \frac{Wz}{2\alpha} - \frac{W^2 t}{4\alpha} \right]$$

**The boundary and initial conditions change also.**

**The b.c. of first kind remains of the 1<sup>st</sup> kind.**

**The b.c. of second kind is changed to the 3<sup>rd</sup> kind.**

**The b.c. of third kind remains the 3<sup>rd</sup> kind.**

## **EXAMPLES OF GF WITH FLOW**

**General GF equation for long-time type for flow in the x-direction is**

$$G_{XUIJ}(x, t | x', \tau) = \frac{X_0(x)X_0(x')}{N_0} e^{\frac{Pe_x x-x'}{2} \frac{\alpha(t-\tau)}{L} + \left( \beta_0^2 - \left( \frac{Pe_x}{2} \right)^2 \right) \frac{\alpha(t-\tau)}{L^2}} + e^{\frac{Pe_x x-x'}{2} \frac{\alpha(t-\tau)}{L}} \sum_{m=1}^{\infty} e^{-R_m^2 \frac{\alpha(t-\tau)}{L^2}} \frac{X_m(x)X_m(x')}{N_m}$$

**where**

$$Pe_x \equiv \frac{UL}{\alpha}, \quad R_m^2 \equiv \beta_m^2 + \left( \frac{Pe_x}{2} \right)^2$$

**The above equation is for  $T$ , not transformed variable  $W$ .**

**XU11**

**Short time form**

$$G_{XU11}^S(x, t | x', \tau) \approx e^{\frac{Pe_x x - x'}{2L}} e^{-\left(\frac{Pe_x}{2}\right)^2 \frac{\alpha(t-\tau)}{L^2}} \left[ \begin{array}{l} K(x-x') - K(x+x') - \\ K(2L-x-x') + K(2L-x+x') + K(2L+x-x') \end{array} \right]$$

where

$$K(z) \equiv \frac{1}{(4\pi\alpha u)^{1/2}} e^{-\frac{z^2}{4\alpha u}}, \quad u \equiv t - \tau$$

$$H_0(z) \equiv e^{-\frac{Pe_x z}{2L} + \frac{Pe_x^2 \alpha u}{4L^2}} \operatorname{erfc} \left( \frac{z}{L} \left[ \frac{4\alpha u}{L^2} \right]^{-1/2} - \frac{Pe_x}{2} \left[ \frac{\alpha u}{L^2} \right]^{1/2} \right)$$

We need derivatives with respect to  $x'$ ; one is

$$\frac{\partial G_{XU11}^S(x, t | 0, \tau)}{\partial n'} \approx \frac{1}{\alpha u} e^{\frac{Pe_x x}{2L}} e^{-\left(\frac{Pe_x}{2}\right)^2 \frac{\alpha u}{L^2}} [xK(x) - (2L-x)K(2L-x) + (2L+x)K(2L+x)]$$

Long time form

$$G_{XU11}(x, t | x', \tau) = \frac{2}{L} e^{\frac{Pe_x x - x'}{2L}} \sum_{m=1}^{\infty} e^{-R_m^2 \frac{\alpha(t-\tau)}{L^2}} \sin\left(m\pi \frac{x}{L}\right) \sin\left(m\pi \frac{x'}{L}\right)$$

## XU22

### Short time form

$$G_{XU22}^S(x, t | x', \tau) \approx e^{\frac{Pe_x x - x'}{2L}} e^{-\left(\frac{Pe_x}{2}\right)^2 \frac{\alpha(t-\tau)}{L^2}} \left[ \begin{array}{l} K(x-x') + K(x+x') + K(2L-x-x') + \\ K(2L-x+x') + K(2L+x-x') + K(2L+x+x') + \\ \frac{1}{L} \frac{Pe_x}{2} [H_0(x+x') - H_L(2L-x-x')] \end{array} \right]$$

## Long time form

$$G_{XU22}(x,t|x',\tau) = \frac{Pe_x}{L} \frac{e^{-Pe_x \frac{x'}{L}}}{1 - e^{-Pe_x}}$$

$$\frac{2}{L} e^{\frac{Pe_x x - x'}{2L}} \sum_{m=1}^{\infty} e^{-R_m^2 \frac{\alpha(t-\tau)}{L^2}} \frac{\left[ m\pi \cos\left(m\pi \frac{x}{L}\right) - \frac{Pe_x}{2} \sin\left(m\pi \frac{x}{L}\right) \right] \left[ m\pi \cos\left(m\pi \frac{x'}{L}\right) - \frac{Pe_x}{2} \sin\left(m\pi \frac{x'}{L}\right) \right]}{R_m^2}$$

# TWO LAYER PARALLELEPIPED

A. Haji-Sheikh, David Yen

**For  $0 < x < A, 0 < y < B, 0 < z < D$**

$$k_1 \left[ \frac{\partial^2 T_1}{\partial x^2} + \frac{\partial^2 T_1}{\partial y^2} + \frac{\partial^2 T_1}{\partial z^2} \right] + g_1 = C_1 \frac{\partial T_1}{\partial t}$$

**For  $0 < x < A, B < y < C, 0 < z < D$**

$$k_2 \left[ \frac{\partial^2 T_2}{\partial x^2} + \frac{\partial^2 T_2}{\partial y^2} + \frac{\partial^2 T_2}{\partial z^2} \right] + g_2 = C_2 \frac{\partial T_2}{\partial t}$$

## **BOUNDARY CONDITIONS**

**At  $x = 0$  &  $A$ ,  $z = 0$  &  $C$ : 1<sup>st</sup> and 2<sup>nd</sup> kinds**

**At  $y = 0$  &  $C$ : 1<sup>st</sup>, 2<sup>nd</sup>, and 3<sup>rd</sup> kinds**

## **INTERFACE RESISTANCE**

## **EIGENVALUES (Only long time form available)**

**Nine different conditions, ( $X_{11}$ ,  $X_{12}$ , etc.)**

**Some are imaginary; some are close to each other.**

$$G_{ij}(x, y, z, t | x', y', z', \tau) = \sum_{p=1}^{\infty} \sum_{m=1}^{\infty} \sum_{n=1}^{\infty} \frac{C_j X_m(x) X_m(x') Z_n(z) Z_n(z') Y_{i,mnp}(y) Y_{i,mnp}(y')}{N_{x,m} N_{z,n} N_{y,mnp}} e^{-\lambda_{mnp}^2(t-\tau)}$$

***i* and *j* = 1 and 2.**

**(Not independent product of the 3 components)**

**Reference: A. Haji-Sheikh and J.V. Beck, Int. J. of Heat and Mass Transfer, Vol. 45, (2002) p. 1865-1877)**

## **IMPLEMENTATION TO CALCULATE $T$ AND HEAT FLUX**

- 1. Time partitioning: Use both short and long time GF in same problem. Needs numerical integration.**
- 2. Spatial partitioning: Use small part volume in one temporal/spatial domain, then a larger one, etc. Avoids need of numerical integration over time.**
- 3. Unsteady surface element method: For connecting two basic and different geometries**

## COMPUTER PROGRAMS

**COND3D**. Parallelepiped, homogeneous body. B.C. of the 1<sup>st</sup>, 2<sup>nd</sup> and 3<sup>rd</sup> kinds on all six boundaries.

Nonzero initial temperature. Vol. Energy Generation.

Uniform conditions over a surface or volume, zero or constant. Many possible cases.

Highly accurate, to 1 part in  $10^{10}$  of maximum value.

Internal verification. Should get the “same” as the partition time is varied.

**EXAMPLE.** Parallelepiped,  $L = 0.1\text{m}$ ,  $W = 0.05\text{m}$ ,  $H = 0.025\text{m}$  at  $x = 0.075\text{m}$ ,  $y = 0.0125\text{m}$ ,  $z = 0\text{m}$

$$x = 0: q = 3500 \text{ W/m}^2; x = L: T = 1000^\circ\text{C}$$

$$y = 0: q = 0 \text{ W/m}^2; y = W: T_\infty = 25^\circ\text{C}, \text{ Ht. Trans. Coef } 60 \text{ W/m}^2\cdot\text{C}$$

$$z = 0: T_\infty = 50^\circ\text{C}, \text{ Ht. Trans. Coef } 10 \text{ W/m}^2\cdot\text{C}; z = H: T_\infty = 40^\circ\text{C}, \text{ Ht. Trans. Coef } 5 \text{ W/m}^2\cdot\text{C}$$

$$k = 0.4 \text{ W/m}\cdot\text{C}; C = 3000000 \text{ J/m}^3\cdot\text{C}; t = 1000\text{s}$$

$$T_0 = 100^\circ\text{C}; \text{ Vol. Energy Gen.} = 135300 \text{ W/m}^3$$

**COND3D RESULTS.** Same to 10 sign. figures for  $t_p = 0.025, 0.05$ . This provides VERIFICATION.

Temperature	x-heat flux	y-heat flux	z-heat flux
208.7524786	-4261.888204	60.6443709	-1587.524786

These values agree with a similar, but not identical, program to all 10 sign. digits, except one with a 5 instead of 4 in the last digit.

**CONSIDER STEADY STATE FOR SAME PROBLEM, time = 10,000,000 s**

**COND3D (TRANSIENT PROGRAM)**

TEMPERATURE = 447.994631779662

HEAT FLUX (X) = -4146.97213360952

HEAT FLUX (Y) = 775.803316289508

HEAT FLUX (Z) = -3979.94631779662

**VERIFSS (KEVIN COLE STEADY STATE)**

The temperature is 447.99463177966

The flux is -4146.9721336095      775.80331628950      -  
3979.9463177966

Agree to within about 13 or 14 digits. Completely different programs.

## **SUMMARY**

- **Digital databases now possible**
- **Verification.**
  - Internal verification**
  - Extreme accuracy possible**
- **Green's functions in heat conduction given**
- **Prototype of database given**